

**AMENDMENTS TO THE CLAIMS**

**Please amend claims 1-12 and cancel claims 13-14 as follows.**

1. (Currently Amended) A power tool, comprising:

a tool bit,

a tool body to which the tool bit is coupled,

an actuating mechanism disposed in the tool body to drive that drives the tool bit linearly by means of pressure fluctuations so as to cause the tool bit to perform a predetermined operation, wherein the actuating mechanism has a driving motor, a motion converting mechanism that converts a rotating output of the driving motor to a linear motion, a piston linearly reciprocating in a longitudinal direction of the tool bit via the motion converting mechanism, a striker disposed in front of the piston to cause the tool bit a linear motion, a first chamber between the striker and the piston, and a second chamber disposed adjacent to the piston within the tool body in an opposite side of the first chamber, and

a dynamic vibration reducer having a weight that reciprocates under a biasing force of an elastic element to reduce vibration of the actuating mechanism, the weight being driven by means of pressure fluctuations caused in the second chamber when the piston reciprocates actuating mechanism.

2. (Currently Amended) The power tool as defined in claim 1, wherein the motion converting mechanism comprises actuating mechanism includes a driving motor, a striker that reciprocates in the axial direction of the tool bit so as to cause the tool bit to linearly move and a crank mechanism that drives the striker by converting a rotating output of the driving motor to a linear motion in the an axial direction of the hammer tool bit, and the second chamber is defined by a crank chamber that houses the crank mechanism the dynamic vibration reducer having a body that houses the weight, wherein the fluctuating pressure caused within the crank chamber by an operation of the crank mechanism is introduced into the body of the dynamic vibration reducer, so that the weight is driven in the direction opposite to the reciprocating direction of the striker.

3. (Currently Amended) The power tool as defined in claim 1, wherein, under loaded driving conditions, in which a load associated with the predetermined power tool operation is applied to the tool bit, the weight is allowed to be driven by means of fluctuating pressure developed in the ~~actuating mechanism~~ second chamber, while, under unloaded driving conditions, in which a load associated with the predetermined power tool operation is not applied to the tool bit, the weight is prevented from being driven by means of fluctuating pressure developed in the ~~actuating mechanism~~ second chamber.

4. (Currently Amended) The power tool as defined in claim 1, wherein, under loaded driving conditions, in which a load associated with the predetermined power tool operation is applied to the tool bit, the weight is allowed to be driven by means of fluctuating pressure developed in the ~~actuating mechanism~~ second chamber, while, under unloaded driving conditions, in which a load associated with the predetermined power tool operation is not applied to the tool bit, the weight is prevented from being driven by means of fluctuating pressure developed in the ~~actuating mechanism~~ second chamber and,

wherein the dynamic vibration reducer includes a first actuating chamber and a second actuating chamber that are defined on ~~both~~ opposite sides of the weight within the body, and wherein, at least under the loaded driving conditions, the fluctuating pressure developed in the ~~actuating mechanism~~ second chamber is introduced into the first actuating chamber, and the second actuating chamber can communicate with the outside.

5. (Currently Amended) The power tool as defined in claim 1, wherein, under loaded driving conditions, in which a load associated with the predetermined power tool operation is applied to the tool bit, the weight is allowed to be driven by means of fluctuating pressure developed in the ~~actuating mechanism~~ second chamber, while, under unloaded driving conditions, in which a load associated with the predetermined power tool operation is not applied to the tool bit, the weight is prevented from being driven by means of fluctuating pressure developed in the ~~actuating mechanism~~ second chamber and the fluctuating pressure developed in the ~~actuating mechanism~~ second chamber is released to the outside of the power tool under the unloaded driving conditions by communicating the second chamber to the outside.

6. (Currently Amended) The power tool as defined in claim 1, wherein the tool bit comprises a hammer bit that performs a predetermined hammer operation by applying a linear impact force to the a work piece, ~~and wherein the actuating mechanism comprises a driving motor, a crank mechanism that is housed in the crank chamber and converts a rotating output of the driving motor into linear motion, a piston cylinder mechanism that is driven by the crank mechanism, and a striker that reciprocates in the axial direction of the hammer bit by the action of an air spring which is caused by relative movement of the piston cylinder mechanism.~~

7. (Currently Amended) The power tool as defined in claim 1, wherein the actuating mechanism includes a piston and a cylinder that slide relative to each other in the an axial direction of the tool bit, wherein the tool bit reciprocates in its axial direction by the action of an air spring which is caused by relative movement of the piston and the cylinder, and wherein the weight is disposed along the a circumferential surface of the cylinder and can slide in the axial direction of the tool bit.

8. (Currently Amended) The A power tool as defined in claim 1, further comprising:  
a driving motor;  
a tool bit;  
a tool holder that holds the tool bit within the power tool;  
a striker that reciprocates in the axial direction of the tool bit so as to cause the tool bit to perform a predetermined operation;  
a cylinder that houses the striker such that the striker can slidingly reciprocate reciprocates within the cylinder,  
a crank chamber;  
an actuating mechanism that is disposed within the crank chamber and drives the striker by converting a rotating output of the driving motor to linear motion, and  
a dynamic vibration reducer that reduces vibration during the operation of the power tool, the dynamic vibration reducer having a weight and a body that houses the weight, wherein the weight can reciprocate under a biasing force of an elastic element and is driven by pressure fluctuations which are caused within the crank chamber when the actuating mechanism is driven, and wherein the fluctuating pressure caused within the crank chamber by driving of the actuating

~~mechanism is introduced into the body of the dynamic vibration reducer, so that the weight is driven in the direction opposite to the reciprocating direction of the striker,~~

wherein the cylinder ~~can move~~ moves between a first position near the tool holder and a second position remote from tool holder than the first position, and under loaded driving conditions in which a load associated with the predetermined operation is applied to the tool bit, the cylinder moves to the second position so as to allow the weight to be driven by means of fluctuating pressure within the ~~crank~~ second chamber, while, under unloaded driving conditions in which a load associated with the predetermined operation is not applied to the tool bit, the cylinder moves to the first position so as to prevent the weight from being driven by means of fluctuating pressure within the ~~crank~~ second chamber.

9. (Currently Amended) The power tool as defined in claim 8, wherein ~~the cylinder has an air spring chamber that causes the striker to reciprocate by the action of an air spring when the actuating mechanism is driven, and wherein~~ under the loaded driving conditions, the cylinder moves to the second position so as to allow the striker to be driven by the action of the air spring function of the ~~air spring~~ first chamber, while, under unloaded driving conditions, the cylinder moves to the first position, so as to prevent the striker from being driven by the action of the air spring function of the ~~air spring~~ first chamber.

10. (Currently Amended) The power tool as defined in claim 9, wherein under the loaded driving conditions, the weight is allowed to be driven by fluctuating pressure within the ~~crank~~ second chamber ~~with a time delay after the striker is allowed to be driven by the action of the air spring function of the~~ air spring first chamber.

11. (Currently Amended) The power tool as defined in claim 8, further comprising an air vent that can communicate the ~~crank~~ second chamber with the outside, wherein when the cylinder moves to the second position, the air vent is closed so as to allow the weight to be driven, and when the cylinder moves to the first position, the air vent is opened so as to prevent the weight to be driven.

12. (Currently Amended) The power tool as defined in claim 8, further comprising an air vent that can communicate the ~~air spring~~ first chamber with the outside, wherein the air vent is

closed when the cylinder moves to the second position and the air vent is opened when the cylinder moves to the first position.

13-14. (Canceled)

**Please add new claims 15 and 16 as follows.**

15. (New) The power tool as defined in claim 1, wherein the dynamic vibration reducer has a body to house the weight, an elastic element connecting the weight with the body, first and second actuation chambers respectively provided in opposite sides of the weight, the first and second actuation chambers being isolated with each other,

wherein the weight is linearly moved by introducing pressure fluctuations caused in the second chamber in an anti-phased manner with respect to pressure fluctuations caused in the first chamber when the piston reciprocates while the weight being biased by the elastic element.

16. (New) The power tool as defined in claim 1, wherein the pressure fluctuations in the second chamber is anti-phased with respect to pressure fluctuations in the first chamber when the piston reciprocates.